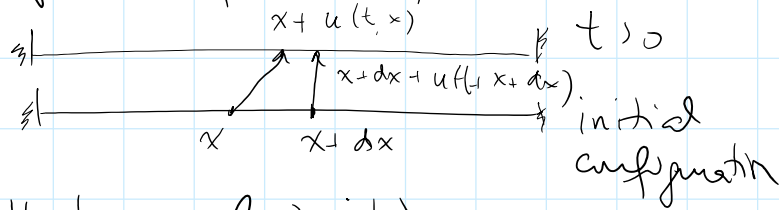


1D elasticity

Lagrangian reference frame



linear constitutive law (Hookean elasticity)

Kinematics

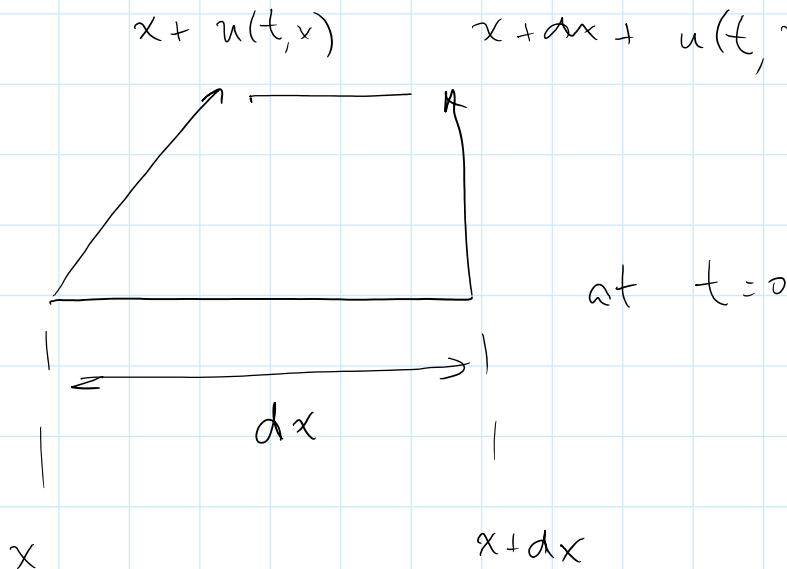
Dynamics

Constitutive relations

Initial configuration

$$u(t=0, x) = 0$$

$u$  = displacement from an initial configuration (assumed to be at static equilibrium)



Kinematics

$$\begin{aligned}
 dl &= x + dx + u(t, x + dx) - (x + u(t, x)) \\
 &= \frac{dx}{1} + \frac{u(t, x + dx) - u(t, x)}{dx} dx \\
 &= [1 + u_x] dx
 \end{aligned}$$

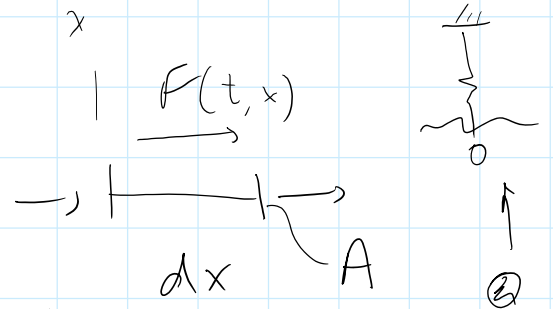
$$= (1 + u_x) u_x$$

$$J(t, x) = u_x$$

$$\frac{dl}{dx} = 1 + u_x$$

Dynamics

$\lambda$  = linear density



$$\underbrace{(\lambda dx)}_{\text{mass}} u_{tt} \left( t, x + \frac{dx}{2} \right) =$$

$$= \underbrace{F(t, x) dx}_{\text{body force}} + \left[ \sigma(t, x + dx) - \sigma(t, x) \right] A$$

$A$  = cross-sectional area, assumed to be constant

$$\lambda u_{tt} = f + \sigma_x A \quad ; \quad \lambda = \rho A$$

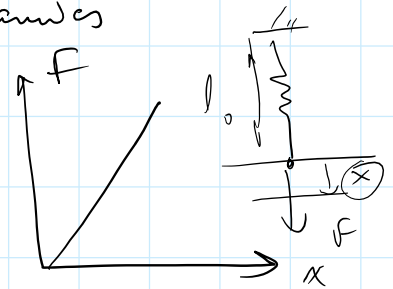
Constitutive law: in general formulate closure between kinematics & dynamics

Hooke's law for a spring

Stress = force per unit area

Strain = relative displacement

$$\sigma = E \varepsilon \quad \sigma = \text{normal stress} = \frac{F}{A}$$



$$\varepsilon = \text{longitudinal strain} = \frac{\Delta l}{l_0} =$$

$$= \frac{\text{displacement}}{\text{initial length}}$$

$$\frac{F}{A} = E \frac{x}{l_0} \Leftrightarrow$$

$$F = \frac{AE}{l_0} x \Rightarrow k = \frac{AE}{l_0}$$

$$\sigma = E \varepsilon$$

$$\lambda u_{tt} = F + \sigma_x A$$

$$dQ = (1 + u_x) dx \Rightarrow \varepsilon = u_x = \frac{u_x dx}{dx}$$

$$\varepsilon_t = u_{xt} = u_{tx} = \sigma_x$$

$$\lambda = \rho A ; F = f A \rho$$

$$\rho \sigma_t = \rho f + \sigma_x \quad \text{dynamics}$$

$$\sigma_x = \varepsilon_t \quad \text{kinematics}$$

$$\sigma = E \varepsilon$$

$$\begin{cases} (I) & \rho \sigma_t - \sigma_x = \rho f \\ (II) & \frac{1}{E} \sigma_t - \sigma_x = 0 \end{cases} \Leftrightarrow \mathcal{L}_t + A \mathcal{L}_x = \psi$$

$$| \quad \mathcal{L} = \begin{pmatrix} \sigma \\ \sigma \end{pmatrix} ; \quad A = \begin{pmatrix} 0 & -E \\ -\frac{1}{E} & 0 \end{pmatrix} ; \quad \psi = \begin{pmatrix} 0 \\ \rho f \end{pmatrix}$$

$$Q = \begin{pmatrix} \rho \\ \sigma \end{pmatrix}; \quad A = \begin{pmatrix} -\frac{1}{\rho} & 0 \\ 0 & \rho \end{pmatrix}; \quad \psi = \begin{pmatrix} p \end{pmatrix}$$

$$\det(A - \lambda I) = 0 \Leftrightarrow \lambda^2 - \frac{E}{\rho} = 0$$

$\lambda_{1,2} = \sqrt{\frac{E}{\rho}} \neq$  P-wave velocity in elastic longitudinal velocity medium

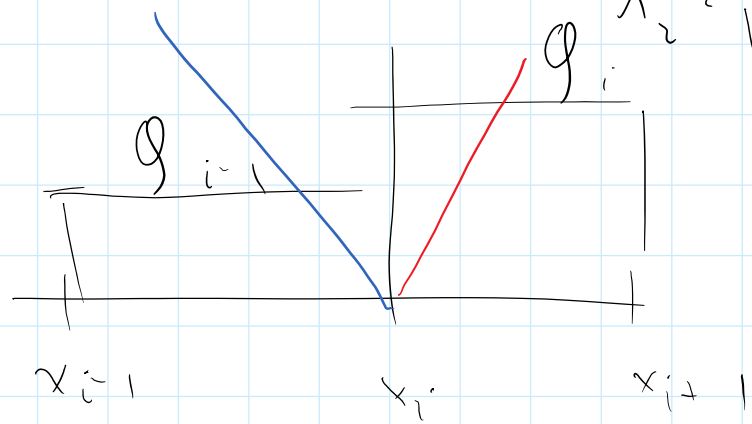
$[\lambda]$  = should be units of velocity

$$= \left[ \frac{N}{m^2} \frac{m^3}{kg} \right]^{1/2} = \left[ \frac{kg \cdot m}{s^2 \cdot m^2} \frac{m^3}{kg} \right]^{1/2} = \frac{m}{s}$$

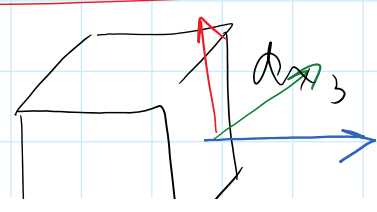
$$\begin{pmatrix} \sqrt{\frac{E}{\rho}} & -E \\ -\frac{1}{\rho} & \sqrt{\frac{E}{\rho}} \end{pmatrix} \Rightarrow \lambda_1 = \sqrt{\frac{E}{\rho}}; \quad r_1 = \begin{pmatrix} 1 \\ c \\ \frac{E}{\rho} \end{pmatrix}$$

$$\sqrt{\frac{E}{\rho}} \cdot 1 - E \cdot \frac{E}{\rho} = 0$$

$$\lambda_2 = -\sqrt{\frac{E}{\rho}}, \quad r_2 = \begin{pmatrix} 1 \\ -c \\ \frac{E}{\rho} \end{pmatrix}$$



Riemann problem



Continuum Mechanics

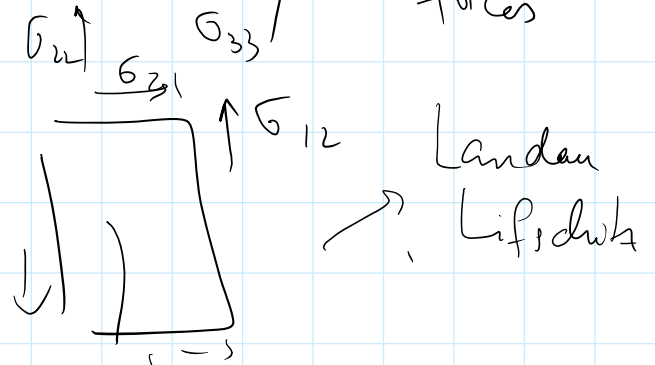
$$\rho \vec{v} dV = \rho f dV +$$



$$\rho \vec{v} dV = \rho f dV + \text{volume distributed forces "body" forces}$$

$$\sigma = \text{stress tensor} = \begin{pmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{pmatrix}$$

$$\sigma = \sigma^T$$



$$\int \frac{dx_1 dx_2 \cdot 1}{m a} \rho = \frac{\sigma_{12} dx_2}{-\sigma_{21} dx_1}$$

$$\vec{I} \cdot \vec{\sigma} = \sum M$$



$$\sigma_{12} \cdot dx_2$$

Fluid

$$\sigma = -p \mathbf{I} + \tau$$

$$p = -\frac{1}{3} \text{Tr } \sigma$$

"isotropic component of normal stress"  
a.k.a. pressure